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## Si70xx HUMIDITY SENSOR DESIGNER'S GUIDE

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### 1. Introduction

This designer's guide describes the Si70xx family of humidity sensor products in a variety of different applications. The first half provides a detailed description of the Si70xx family of humidity sensors including specific application examples. System-level design considerations including sensor placement, system calibration and product use are discussed. Finally, special handling considerations for both the sensor device and the end product containing the sensor are discussed.

The second half begins with a brief description of humidity and the various ways it is quantified. It is important to understand the terminology and the relative merits of each measurement. Next, methods for measuring humidity and measurement challenges are presented. Finally, the impact of humidity on comfort is discussed.

Appendix A contains valuable reference information about industry standards applicable to the measurement and control of humidity. Appendix B contains a description of several equations useful for humidity related calculations. Appendix C contains a convenient reference containing unit conversions, coefficients and a glossary of humidity related terms used in this document. Appendix D explains the nonlinear correction of the Si7013 voltage input.



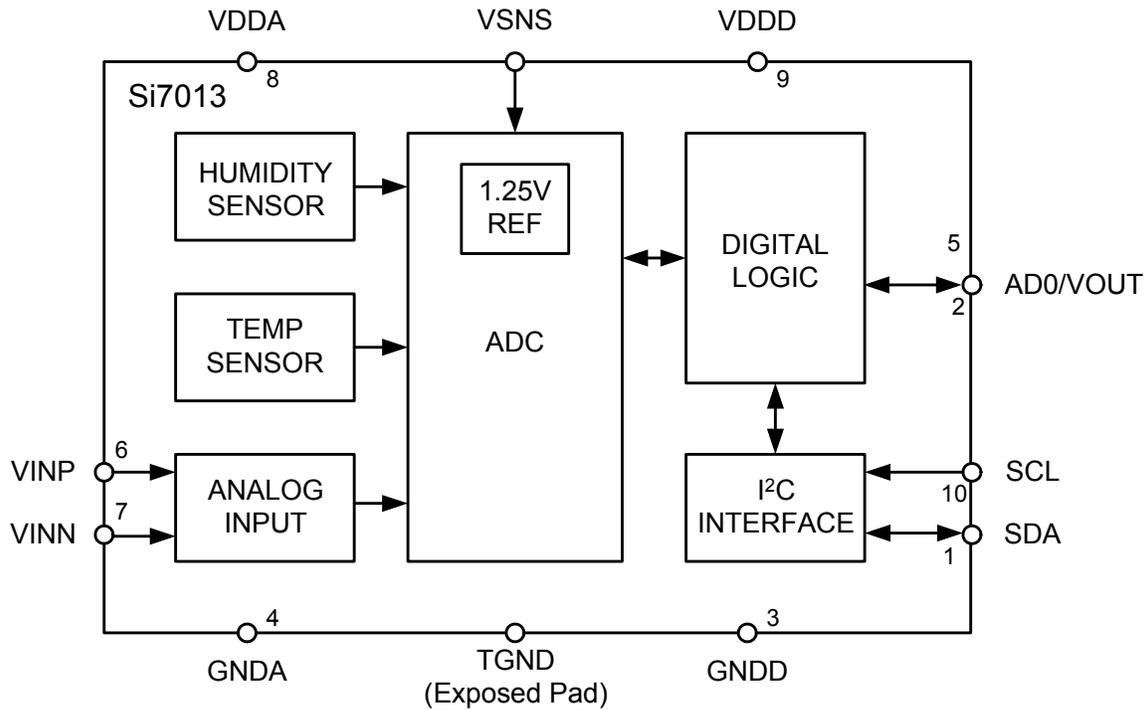
**Figure 1. Si7013 Relative Humidity and Temperature Sensor Shown with and without Hydrophobic Cover Installed**

## 2. Si70xx Monolithic Humidity and Temperature Sensors

The Si70xx family uses state-of-the-art sensing technologies to create precise monolithic humidity and temperature sensors. Temperature is sensed by a precision V<sub>be</sub> referenced circuit on the die. Humidity is sensed by measuring the capacitance change of a low-k dielectric layer applied to the surface of the die. Consequently, both temperature and humidity are precisely measured in extremely close proximity on the same monolithic device, which is critical for accurate dew point measurement.

The Si7005 was the first-generation sensor in a 4 mm x 4 mm package. The Si7015 is a pin-compatible upgrade (see “AN764: Upgrading from the Si7005 to the Si7015”). The Si7013, Si7020, and Si7021 second-generation parts come in an industry-standard 3 mm x 3 mm package, and are collectively referred to as the Si70xx throughout the document.

Figure 2 shows a functional block diagram of the Si7013 humidity and temperature sensor. Very few external components are required. Communication with the device is via the I<sup>2</sup>C bus SDA and SCL pins. The Si7013 has an optional analog input path for measurement of a remote thermistor or any other analog voltage. It also has a dual function pin for I<sup>2</sup>C address selection and thermistor biasing. Depending on the microcontroller used in the application, the 10 kΩ pull-ups on the I<sup>2</sup>C bus may be included in the microcontroller. The only other component required is the 0.1 μF power supply bypass capacitor.

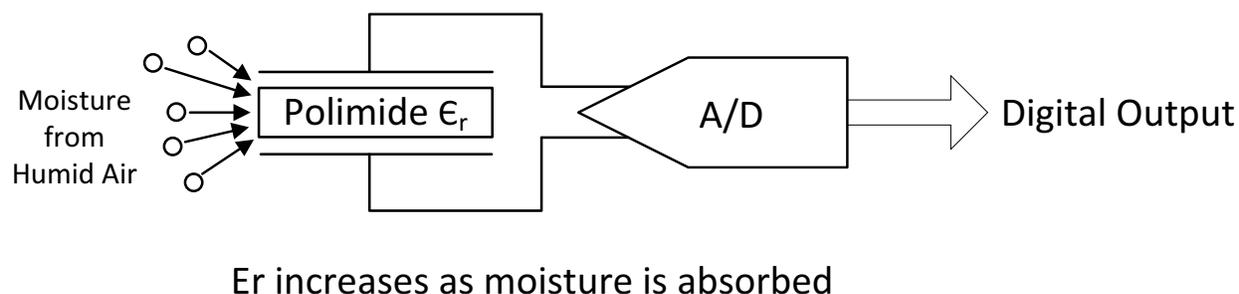


**Figure 2. Si7013 Functional Block Diagram**

Refer to the data sheets (e.g., Si7013) for detailed pin descriptions, register summary, and timing details.

## 2.1. Humidity Sensor Technology

Figure 3 illustrates the humidity sensor configuration. The dielectric layer is exposed to ambient air through the package opening. Humidity in the air is exchanged with the dielectric material until equilibrium is reached. The permittivity of the dielectric material is proportional to the amount of moisture it has absorbed. The capacitance increases as the ambient air becomes more humid and decreases as the ambient air becomes less humid. If the Si70xx is used at the dew point and condensation occurs on the sensor, the on-chip heater can be activated to dry the sensor and restore operation once the sensor is above the dew point.



**Figure 3. Humidity Sensor Configuration**

## 2.2. Temperature Sensor

The Si70xx parts have a very accurate (as good as  $\pm 0.4$  °C) temperature sensor integrated into the device. The Si70xx family of devices is designed to consume very little power and does not artificially heat the temperature sensor unless, of course, the heater is turned on. The thermal paddle on the bottom of the chip is electrically connected to ground inside the chip. The paddle can be soldered to a ground plane on the PCB which can in turn be extended beyond the chip or can be left floating. In either case, the ground pad is the thermal input to the on-chip temperature sensor. Care must be taken to isolate the chip from thermal sources in the system in which it is included. Refer to the section on PCB layout guidelines for more details on PCB layout and system design considerations to maximize sensor performance.

## 3. System Design Considerations

### 3.1. Power Requirements

The Si70xx devices are inherently low power if the heater is not used. For example, Si7013 power consumption during a conversion is 150  $\mu$ A typical, and the power consumption in idle mode is less than 100 nA. This low power consumption means that there is no significant self-heating in the conversion process.

#### 3.1.1. Battery Life Considerations

Normally, battery life is rated in milliamp-hours (mAh). For example, an Energizer<sup>®</sup> E91 battery is rated for approximately 2500 mAh for light loads when allowed to discharge to 0.9 V.

Considering the case of an Si7013, which has a  $V_{DD}$  range of 1.71 to 3.6 V, two AA batteries in series provide 3.0 V when charged and 1.8 V when fully depleted, so this is acceptable.

Again, for the case of an Si7013, a humidity and temperature conversion typically consumes 150  $\mu$ A. In normal mode, the total time for an RH and temperature conversion is 8.4 msec, and, in fast mode, this is reduced to 5.5 msec. A temperature conversion is performed every time an RH conversion is done (to allow for temperature correction).

If a conversion is performed once per second, then  $31.5 \times 10^6$  conversions occur per year. The number of milliamp-hours is as follows:

$$\text{milliamp hours (mAh)} = \text{number of conversions} \times \text{conversion time (msec)} \times \frac{\text{conversion current (mA)}}{(1000\text{msec/sec} \times 3600\text{sec/hour})}$$

Plugging in values, this translates to 11.1 mA-hours for one year in normal mode and 7.2 mA-hours for fast mode.

The 100 nA quiescent current of the Si7013 will consume an additional number of milliamp hours, as shown below:

$$\text{milliamp hours (mAh)} = \text{quiescent current (nA)} \times \frac{8760 \text{ (hours/year)}}{10^6 \text{ nA/mA}}$$

At 100 nA quiescent current, this is 0.876 mA-hours per year.

Even when targeting a 10-year battery life, the sensor itself would consume less than 120 mA hours of charge in normal mode. This means that only about 5% of the available 2500 mAh is consumed by the sensor.

Of course, in a practical system, there are many other drains on a battery, and, often, coin cells, which have much less capacity than AA cells, are used. On the other hand, it is generally acceptable to reduce the conversion rate well below once per second, meaning that a sensor, such as the Si7013, will generally consume a small portion of the available battery life.

## 3.2. Sensor Placement

The humidity sensor should be placed away from any heat sources (including sunlight) and should have direct access to the ambient where RH is being measured. If the optional hydrophobic filter is not used, the Si70xx must be protected from exposure to dust and liquids during soldering and normal use. While the Si70xx devices can provide accurate humidity in still air, some level of airflow around the sensor is recommended to improve response speed and reduce susceptibility to condensation.

Many systems with humidity sensors mount the sensing device in a location facing an opening in the system enclosure to permit exposure to the ambient environment.

A particularly interesting application for humidity sensors is in mobile handsets. To avoid heating from the handset electronics, the RH sensor should be thermally isolated as much as possible. For example, the sensor could be placed in a small plastic chamber that is exposed to the ambient on one side and insulated from the heat-producing phone electronics on the other sides. Most mobile handsets have two or more PCB's in their system design. The RH/temperature sensor can be placed on one of the boards that does not have the high-power wireless radio and SOC devices in order to improve thermal isolation. A dust and liquid filter, such as the hydrophobic filter available from Silicon Labs with the Si70xx devices, will ensure greater accuracy of the sensor over its life. Alternatively, the side of the chamber that is exposed to the ambient can use such a filter to keep dust out of the entire chamber. If it is possible to estimate the thermal rise of the sensor (either by characterization or the use of a separate temperature sensor), the equations in "Estimating RH with Heating" on page 26 can be used to develop a more accurate estimate of the true ambient humidity.

## 3.3. Dealing with Condensation and High Humidity

Condensation will cause erroneous readings if the condensation forms on the polyimide film. The hydrophobic filter prevents liquid water from penetrating so condensation on the outside of the part will generally not result in condensation on the polyimide sensing film of the Si70xx. However, condensation on the PCB can affect reliability and signal integrity.

Prolonged exposure to high humidity will cause gradual drift of the sensor readings. All members of the Si70xx family have on-chip heaters that can be used to heat the chip to counter local high humidity and reduce this drift. Turning on the heater with the control setting 0x3 for Si7013 when high RH is seen heats the sensor about 5 °C (depending on PCB design and airflow), which results in a ~30% drop in local RH. Turning on the heater when high RH (over 70%) is seen will help reduce sensor drift and may aid in keeping condensation from forming on the outside of the part. The heater can be turned off when the RH reading drops below 40%.

However, due to variability in air flow and heater current, the Si70xx heating can vary  $\pm 2$  °C which translates to  $\pm 12\%$  RH making RH readings with the heater on unreliable. Depending on the nature of the application there are several ways of dealing with this:

- If the only need is to sense when RH is high, then using the above algorithm (turning on the heater at 70% RH and turning it off at 40% RH) if the heater is on, the RH is generally over 60%.
- While the RH reading is not accurate, the dew point reading is fairly accurate (although generally about 2 °C low) with the heater on. If dew point is the only concern it can be calculated.
- The air ambient temperature can be sensed with a separate sensor and RH can be calculated using the dew point as above and the air ambient temperature provided by the auxiliary temperature sensor.
- If the amount of heating can be measured or characterized, see "Estimating RH with Heating" on page 26 for the recommended method of calculating RH with some amount of heating.

The Si7013 has options for increasing the heater current up to 94 mA with  $V_{DD} = 3.3$  V. Depending on the PCB layout and thermal design, this will cause as much as 40 °C rise in junction temperature. This amount of heating is enough to drive off condensation or liquids that were absorbed into the filter cover.

## 3.4. PCB Layout

The Si70xx should be thermally isolated from the equipment connected to it to prevent heat from the equipment from affecting RH. The Si70xx should be thermally immersed in the ambient environment it is intended to sense.

One strategy for accomplishing this is to put the Si70xx on a small PCB and run a ribbon cable to the host processor. The small PCB should be placed away from heat sources and should be placed in the ambient environment as much as possible. That said, even with the hydrophobic filter, keeping dust, liquids and cleaning agents away from the sensor is required.

## 3.5. Design and Bring-Up Checklist

- ❑ Be sure the sensor is placed away from heat sources and exposed to the environment it is measuring.
- ❑ Prevent the active area of the sensor from being exposed to liquids, dust, and other contaminants as well as sunlight or other UV sources. The optional filter cover available with Si70xx parts serves this purpose and is compatible with soldering.
- ❑ Generally, avoid the use of ground planes around the Si70xx, which could conduct heat from external sources. Route the ground connection.
- ❑ Do not connect unused pins. Make sure the CSb pin of the Si7015 is low prior to starting I<sup>2</sup>C communications.
- ❑ The Si7005 should not be on the same bus as other I<sup>2</sup>C devices when it is active. It acknowledges data bytes that match its address. This issue has been resolved with other members of the Si70xx family
- ❑ Be sure to meet all of the timing and level requirements of the device. The Si7005 can tolerate SDA or SCL higher than V<sub>DD</sub> and has 8.5 mA drivers. Other members of the Si70xx family have 2.5 mA drivers and do not tolerate I<sup>2</sup>C pins higher than V<sub>DD</sub>.
- ❑ Route the I<sup>2</sup>C signals away from analog nodes and noisy digital nodes.
- ❑ Use 0.1 µF bypass capacitors on V<sub>DD</sub> placed close to the sensor.
- ❑ Pay careful attention to I<sup>2</sup>C protocols, such as start and stop conditions, the repeated start of a read transaction, and proper treatment of the Acknowledge bit.
- ❑ Allow adequate time for initialization (per data sheet).
- ❑ If the optional thermistor sensing of Si7013 is used, make sure the thermistor is thermally isolated. If there are long leads to the thermistor, use a twisted pair. Avoid noise pick up; use either a shield or capacitive filter.

## 3.6. Si70xx Self Test

The following steps define a reliable test of the Si70xx family that uses the integrated heater:

1. Read and write all I<sup>2</sup>C registers checking for expected values and capability of modifying where appropriate
2. Perform an RH and temperature measurement.
3. Turn on the heater and wait 60 seconds.
4. Check for delta temperature with heater on. For Si7015, this is expected to be >3 °C. For Si7021, this is over 0.5 °C, and, for Si7013, this can be adjusted changing the heater setting. A setting of 0x3 will give over 3 °C.
5. Check for delta RH is >RH initial x 4x (delta temperature in °C).

## 3.7. ESD Considerations

It is desirable to expose the Si70xx sensor to the environment. For the sensor to respond to the environment there must be a way for the air being sensed to reach the sensor (the environmental access port). This means that the sensor may also be exposed to ESD as specified in IEC 61000-4-2 with ESD peak voltage of up to  $\pm 15$  kV.

When the cover is not used and the ESD source is directly over the package opening, it is possible for ESD to arc into the sensor area and cause damage. This can be avoided by using of the Silicon Labs filter cover or by placing the Si70xx so that the sensor opening is offset from the environmental access port.

The above approaches will prevent ESD discharge into the sensor area, but ESD discharge to the leads may still be possible. The best practice for ESD protection of the leads is to arrange the sensor placement and environmental access so that high-level ESD events will preferentially be directed to ground (i.e., have an exposed ground trace or ground shield closer to the environmental access port than the sensor). If grounded, a metal case is used; this is also effective for ESD protection.

If it is not possible to protect the leads of the device from ESD, unused leads should be connected to VDD. High-quality ESD protection diodes can be used on leads that have signals on them. The ESD protection device should be rated for more than 15 kV immunity and should limit ESD voltage peaks to less than 10 V. Some examples that have been tested include Vishay MSP3V3 and COMCHIP CPDQ3V3U-HF.

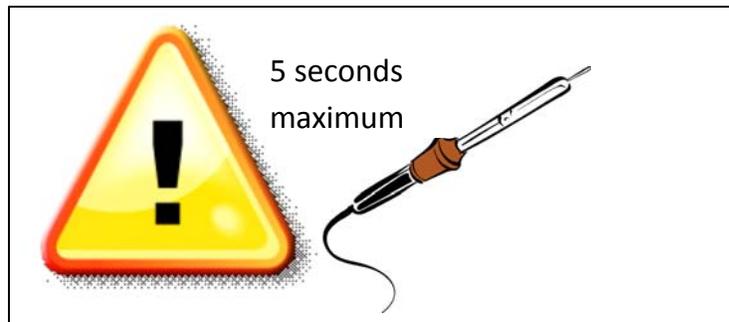
## 4. Humidity and Temperature Sensor Special Handling Considerations

### 4.1. Product Storage

The Si70xx are shipped in sealed anti-static bags. The sensors may be stored in a humidity and temperature controlled (RH: 20% to 60%, Temp 10 °C to 35 °C) environment for up to one year after being removed from the bag prior to assembly (the moisture sensitivity rating is MSL2). Do not store the sensors in polyethylene bags (typically blue, yellow, or pink) because these emit gasses that can affect the sensor. Metallic, anti-static, sealable, moisture-barrier bags are recommended for storage. Do not use sealants or tapes to seal inside the packaging.

### 4.2. Assembly Flow

Limit soldering iron rework to 5 seconds per lead, for complete rework use a new sensor as manual removal and and soldering can shift sensor accuracy outside of data sheet limits. Avoid the use of hot air rework tools.



**Figure 4. Limit Solder Rework to 5 Seconds or Less**

The sensor opening should generally be covered during soldering to prevent flux from getting on the sensor surface. Further it is recommended that the Si70xx only be soldered with standard reflow (no hand soldering or hot air tools). The hydrophobic filter is compatible with standard reflow soldering. If the hydrophobic filter cover is not used, Kapton tape will serve the same purpose, although it has to be removed after soldering. Soldering iron touch-up is possible if liquid flux is not needed and care is taken to avoid excessive heating (5 seconds per lead). If complete rework is needed, the recommended method is to use a new part and reflow the entire board.

The Si70xx filter cover is compatible with standard reflow soldering. It provides lifetime protection against dust and liquids and should not be removed after soldering.



**Figure 5. Do not Remove Si70xx Protective Cover**

The reflow should follow JEDEC standards for lead free solder and reflow with a peak temperature of less than 260 °C. “No clean” solder paste should be used. The use of an ultrasonic bath with alcohol for cleaning after soldering is specifically not recommended. The Si70xx sensor opening must be kept clean and free of particulates during assembly; the pre-installed white filter cover will protect the opening from particulates. Ensure the sensor opening does not come into contact with conformal coatings. Do not expose the Si70xx to volatile organic compounds or solvents. If installed, do not remove the white filter cover from the devices.

The use of water-soluble flux and water rinse after soldering is permissible if done with care. The use of DI water is recommended. If the hydrophobic cover is used, a spray pressure of less than 40 PSI will prevent water entry into the cover. Without the cover, care would need to be taken to avoid particles from the water or from blow drying to contaminate the sensor area.

The high-temperature soldering process will introduce a recoverable shift in the sensor indication. Generally, accuracy will be back within tolerance limits within 48 hours of soldering if the sensor is stored in normal ambient conditions with ~50%RH.

### 4.3. Sensor Sensitivity to Chemicals and Vapors

The Si70xx is sensitive to many chemicals and fumes. Notably, household cleaning agents, such as ammonia, are known to cause sensor readings to drift. To maintain accuracy of the SI70xx, avoid exposure to chemical fumes and contaminants.

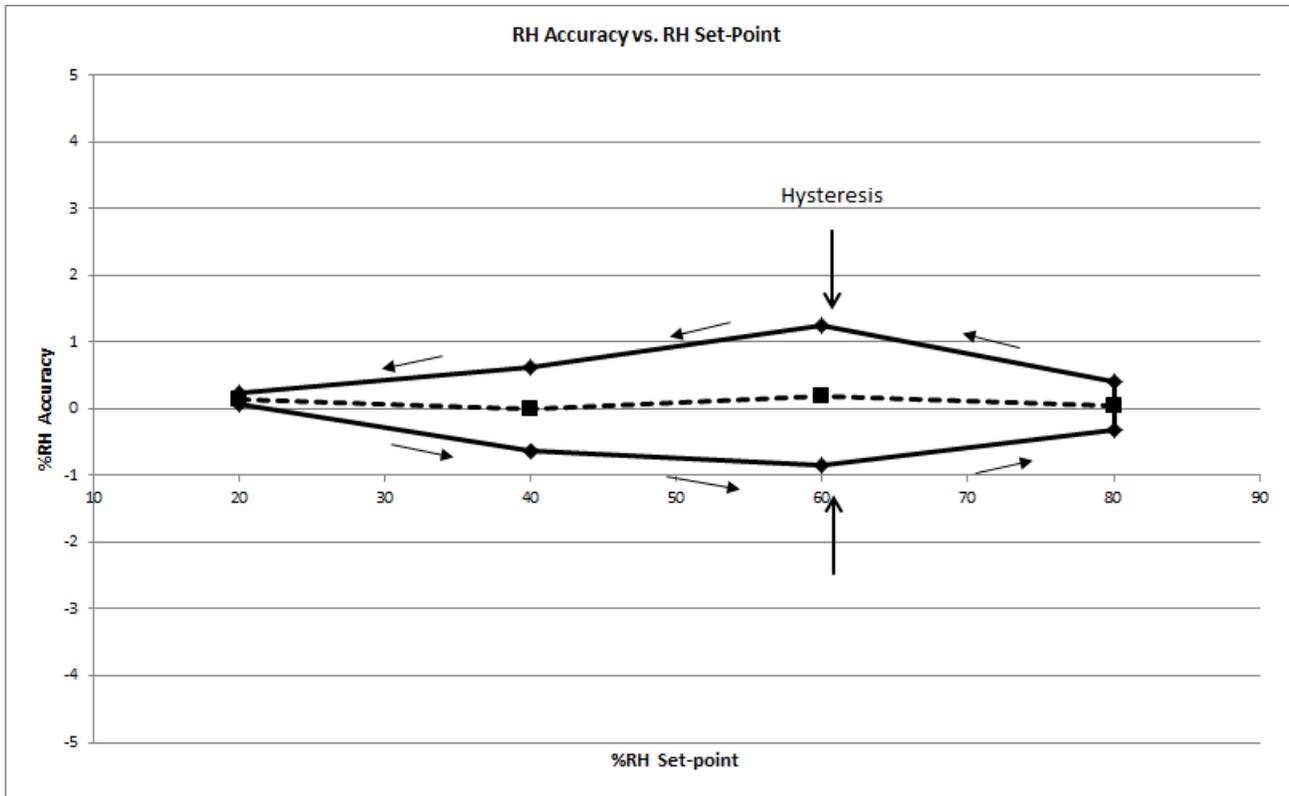
- Inert dust (e.g., talc) is essentially benign.
- Excessive amounts of dust can slow response.
- Contaminants or particles embedded in the polyimide can affect the RH accuracy.
- Certain polyethylene bags will outgas and damage the sensor.
- Bleach, hydrogen peroxide, ammonia, and other chemicals can affect or damage the sensor.

### 4.4. Recovering Calibration after High Humidity or Chemical Exposure

Typically, initial accuracy can be recovered by baking the sensor at 125 °C for 12 hours followed by ~2 days storage period in normal ambient conditions with ~50%RH. High RH exposure (i.e., 75% RH for 12 hours) will accelerate the post-bake recovery, but, after high RH exposure, approximately two days at normal RH is still recommended for the device to fully recover its accuracy.

### 4.5. Relative Humidity Sensor Accuracy

To determine the accuracy of a relative humidity sensor, it is placed in a temperature and humidity controlled chamber. The temperature is set to a convenient fixed value (typically 25–30 °C) and the relative humidity is swept from 20 to 80% and back to 20% in the following steps: 20% – 40% – 60% – 80% – 80% – 60% – 40% – 20%. At each set-point, the chamber is allowed to settle for a period of 30 minutes before a reading is taken from the sensor. Prior to the sweep, the device is allowed to stabilize to 50%RH. The solid trace in Figure 6, “Measuring Sensor Accuracy Including Hysteresis,” shows the result of a typical sweep.



**Figure 6. Measuring Sensor Accuracy Including Hysteresis**

The RH accuracy is defined as the dotted line shown in Figure 6, which is the average of the two data points at each relative humidity set-point. In this case, the sensor shows an accuracy of 0.25%RH. The Si70xx accuracy specification includes:

- Unit-to-unit and lot-to-lot variation
- Accuracy of factory calibration
- Margin for shifts that can occur during solder reflow

The accuracy specification does not include:

- Hysteresis (typically  $\pm 1\%$ )
- Effects from long term exposure to very humid conditions
- Contamination of the sensor by particulates, chemicals, etc.
- Other aging related shifts ("Long-term stability")
- Variations due to temperature. RH readings will typically vary with temperature by less than  $\pm 0.05\% \text{ } ^\circ \text{C}$ .

## 4.6. Hysteresis

The moisture absorbent film (polymeric dielectric) of the humidity sensor will carry a memory of its exposure history, particularly its recent or extreme exposure history. A sensor exposed to relatively low humidity will carry a negative offset relative to the factory calibration, and a sensor exposed to relatively high humidity will carry a positive offset relative to the factory calibration. This factor causes a hysteresis effect illustrated by the solid trace in Figure 6. The hysteresis value is the difference in %RH between the maximum absolute error on the decreasing humidity ramp and the maximum absolute error on the increasing humidity ramp at a single relative humidity setpoint and is expressed as a bipolar quantity relative to the average error (dashed trace). In the example of Figure 6, the measurement uncertainty due to the hysteresis effect is  $\pm 1.0\%RH$ .

## 4.7. Prolonged Exposure to High Humidity

Prolonged exposure to high humidity will result in a gradual upward drift of the RH reading. The shift in sensor reading resulting from this drift will generally disappear slowly under normal ambient conditions. The amount of shift is proportional to the magnitude of relative humidity and the length of exposure. In the case of lengthy exposure to high humidity, some of the resulting shift may persist indefinitely under typical conditions. It is generally possible to substantially reverse this effect by baking the device as described in the following section.

## 4.8. Bake/Hydrate Procedure

After exposure to extremes of temperature and/or humidity for prolonged periods, the polymer sensor film can become either very dry or very wet, in each case the result is either high or low relative humidity readings. Under normal operating conditions, the induced error will diminish over time. From a very dry condition, such as after shipment and soldering, the error will diminish over a few days at typical controlled ambient conditions, e.g., 48 hours of  $45 \leq \%RH \leq 55$ . However, from a very wet condition, recovery may take significantly longer. To accelerate recovery from a wet condition, a bake and hydrate cycle can be implemented. This operation consists of the following steps:

- Baking the sensor at 125 °C for  $\geq 12$  hours
- Hydration at 30 °C in 75% RH for  $\geq 10$  hours

Following this cycle, the sensor will return to normal operation in typical ambient conditions after 48 hours.

## 5. An Introduction to Humidity

Atmospheric air normally contains water vapor and can be thought of as a mixture of ideal gasses. Dry air (no moisture content) is the combination of approximately (on a mole basis) 78.09% N<sub>2</sub>, 20.95% O<sub>2</sub>, 0.93% Ar and 0.03% CO<sub>2</sub> and trace elements. The amount of water vapor found in air depends on available liquid water (or ice), temperature, pressure and ranges from nearly zero to the point of saturation called “dew point (frost point)”. Water vapor enters the air by evaporation due to the vapor pressure of water or ice.

### 5.1. Vapor Pressure

There are a few key concepts to keep in mind when discussing vapor pressure. Pure water vapor pressure,  $p$ , is due to water vapor pressure over water or ice without the presence of other gases such as air. In combination with air, the actual water vapor pressure is increased by a dimensionless factor referred to as a water vapor enhancement factor,  $f$ . This factor is a weak function of temperature and pressure and is approximately 0.47% at sea level and 20 °C. The actual vapor pressure of water vapor,  $p'$ , is the pure water vapor pressure,  $p$ , multiplied by the enhancement factor,  $f$ .

$$p' = pf = p(1.0047)$$

Dalton's law states that the total pressure of a mixture of gasses is equal to the sum of the partial pressures of each component gas and assumes the combination of gasses behaves like an ideal gas. In an ideal gas mixture such as air, the total pressure is the sum of the partial pressures of each gas. The pressure of air at any point can be calculated as follows.

$$P = P_{N_2} + P_{O_2} + P_{H_2O} + P_{Ar} + P_{CO_2}$$

Note that  $p_{H_2O}$  is the actual vapor pressure of water in air referred to as  $p'$ .

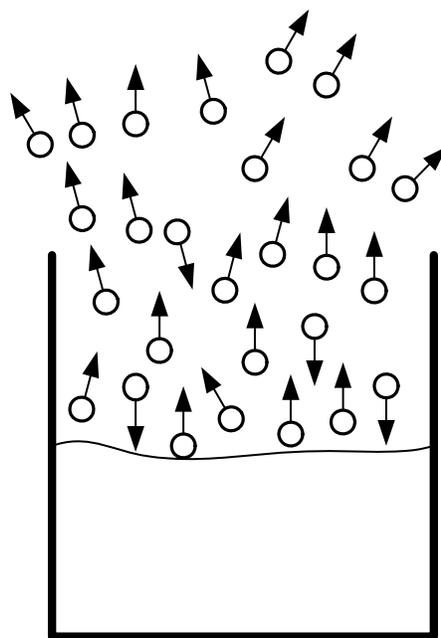
There are several useful equations when working with vapor pressure. The best one to use depends on the available information you have about the problem you need to solve, the range of conditions for the problem the degree of accuracy required and the computational resources available. These equations and their range of application are discussed in detail in appendix B. Pressures referred to in this document are absolute (not gauge) unless otherwise specified.

### 5.2. Temperature

The relative humidity value can change significantly with even slight variations in temperature. For example, a 1 °C change in temperature at 35 °C and 75% relative humidity will introduce a -4% RH change. A higher temperature increases the ability of air to absorb moisture and a lower temperature decreases the ability of air to absorb moisture. Temperature changes can introduce moisture variations in an air mass if condensation occurs or through secondary impacts such as changing the moisture absorption or desorption of environmental materials in an enclosure. For humidity sensors that respond in proportion to relative humidity and not absolute humidity, the issue of temperature measurement error is not significant unless conversion to dew point, absolute humidity or any other measurement of water vapor in the air is required. In the case of a dew point calculation, a 1 °C error in the measurement of the temperature will produce approximately a 1 °C error in the calculation of the dew point. This temperature dependency not only emphasizes the importance of accurate temperature measurement, it also highlights the necessity of thermal stability, which can be difficult to achieve. Even if the temperature and humidity measurements are taken in relatively close proximity, there can be considerable differences in corresponding levels of humidity and temperature. To achieve the most accurate measurement it is best if the humidity and temperature measurements are taken as close as possible to each other—ideally co-located on the same chip.

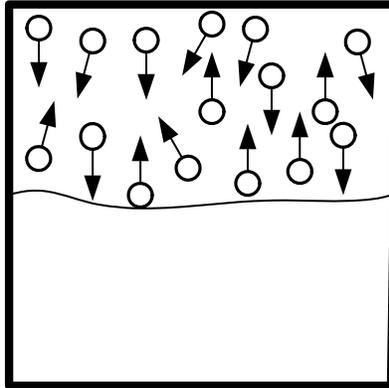
### 5.3. Evaporation

The average energy of molecules in a liquid is directly proportional to temperature. The vapor pressure of water will increase as it is heated. When it reaches 100 °C at sea level, the vapor pressure will equal the atmospheric pressure (approximately 760 Torr or 101.325 kPa) and the water will boil. The energy (vapor pressure) of water molecules in an open partially filled container of liquid water (or ice), Figure 7, is such that some of the molecules have enough energy to escape the attractive forces holding the water together and evaporate into the atmosphere. Eventually all of the water will evaporate assuming the air is not saturated. If the container is sealed, Figure 8, water molecules will evaporate into the air space above the water and some of the evaporated water molecules will condense back into the liquid water. The rate of evaporation will exceed the rate of condensation until the air above the water becomes saturated with water vapor. Once the air is saturated the rates of evaporation and condensation will be equal (the vapor pressure will equal the saturation pressure inside the closed container). Evaporation takes place at the surface of the liquid while boiling can take place throughout the volume of the liquid.



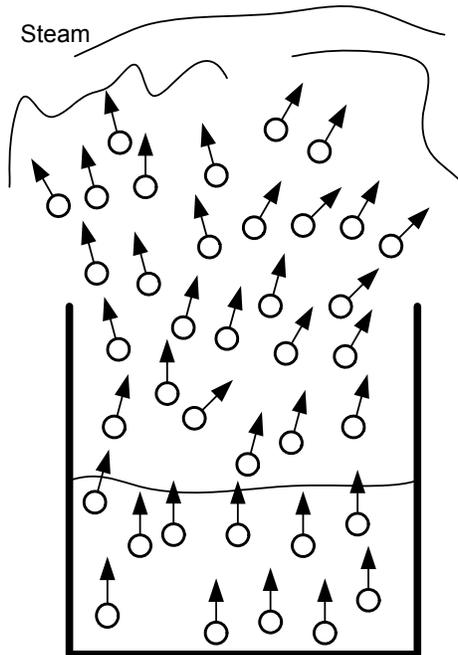
Evaporation > Condensation  
Humidity in the open air is increasing

**Figure 7. Water in Open Container**



Evaporation = Condensation  
Saturated vapor

**Figure 8. Water in Closed Container**



$p' > P$   
Evaporation >>> Condensation  
Humidity in the open air is increasing

**Figure 9. Water Boiling**

Boiling starts at the liquid surface when the vapor pressure equals the atmospheric pressure. As temperature increases, bubbles form below the surface as the vapor pressure increases to equal the atmospheric pressure plus the additional pressure due to the weight of the column of liquid above the bubble. As temperature and vapor pressure continue to increase, bubbles will form throughout the entire volume of liquid as shown in Figure 9. Water vapor is completely absorbed into the air which can cause the air in close proximity to the boiling water to become locally saturated. Visible steam consists of very small water droplets which have condensed out of this localized super-saturated air.

## 6. How Humidity is Quantified

Humidity represents the amount of water vapor contained in the air and can be quantified in many ways. Several of the terms describing humidity are defined differently for meteorology applications and thermodynamics or chemical engineering applications. For this reason, it is important to understand the context of the application. The following is a brief description of the most common terms for quantifying humidity.

### 6.1. Absolute Humidity

Absolute humidity in the context of meteorological applications, sometimes referred to as “volumetric humidity”, is defined as the mass of water vapor dissolved in a total volume of moist air at a given temperature and pressure. Typical units are g/m<sup>3</sup> or kg/m<sup>3</sup>. The value of absolute humidity defined in this manner changes with temperature and pressure and is inconvenient to use in many engineering applications.

Absolute humidity for use in thermodynamics or chemical engineering applications is defined as the ratio of water vapor mass to dry air mass. Typical units are kg/kg. Other names for this ratio include mass mixing ratio, humidity ratio, mass fraction or mixing ratio. This quantity is simpler to use and more accurate for mass balance or heat calculations. Due to the conflicting definitions of absolute humidity, caution is required when using this term.

### 6.2. Specific Humidity

Specific humidity,  $Y_w$ , can be defined for meteorology applications as the ratio of water vapor mass per mass of moist air expressed as g/kg or kg/kg. Specific humidity is constant with changes in temperature and pressure for conditions above the dew point and is a useful quantity in meteorology. The rate of evaporation of water is directly proportional to specific humidity.

### 6.3. Relative Humidity

Relative humidity is the ratio of actual water vapor present in air with the amount of water vapor that would be present in air at saturation, expressed as a percentage. The official symbol for relative humidity is  $\Psi$  although RH, %RH, rh, or %rh are commonly used. Relative humidity can be expressed as the ratio of the actual vapor pressure,  $p'$ , to the saturation vapor pressure,  $p_s'$ , at a constant temperature over a plane of liquid water.

$$RH (\%) = p' / (p_s') \times 100$$

### 6.4. Dew Point

Upon heating, the capacity of air to absorb moisture increases. Consequently, the relative humidity of air decreases as the air is heated. Conversely, as moist air is cooled, its capacity to absorb moisture decreases and relative humidity increases. The dew point is the temperature, assuming constant pressure, moist air is saturated (reaches 100% relative humidity) and cannot absorb additional water. As the temperature is decreased past the dew point, moisture condenses until the air is saturated (reaches 100% relative humidity) at the new lower temperature.

### 6.5. Frost Point

Frost point is the same as dew point over solid ice where the condensate is frost instead of liquid water.

## 7. Humidity Measurement

For relative humidity measurements, it is not necessary to measure the ambient temperature unless you are using a psychrometer. However, to determine the dew point or absolute humidity, the ambient air temperature is required. Accurate air temperature measurements can be a significant challenge, since air is a poor thermal conductor and the temperature at any given point can be impacted by air currents and temperature gradients. It is very important to understand the dynamics of your measurement system and the dynamics of the environment being measured. Before taking a measurement, you always need to wait long enough to ensure the temperature and humidity are stable and the sensor(s) of the measurement instrument are in equilibrium with the ambient conditions to be measured.

A wide range of techniques are employed to measure humidity. These range from simple mechanical indicators, to highly complex and expensive analytical instruments. In general, measuring humidity (dew point, absolute humidity, specific humidity, mixing ratio, relative humidity or equivalent wet bulb temperature) is not a trivial task. Many of the instruments currently available have poor accuracy, narrow-bandwidth, contamination issues, hysteresis and measurement drift over temperature and time. Regular calibration is required on some instruments, which is both inconvenient and expensive. Some instruments are large, awkward, and expensive. Discussed below, are different methods of humidity sensing. The new generation of humidity sensor technology used in Silicon Labs' solid state humidity sensors provide superior accuracy, minimum drift, low cost, low power, small size and ease of use.

### 7.1. Psychrometer

A psychrometer is the oldest method for measuring humidity more commonly known as the wet bulb/dry bulb method. A psychrometer consists of two thermometers, one with an ordinary dry bulb and the other with a moist cloth covering the bulb (wet bulb). Evaporation from the moist cloth lowers the wet bulb thermometer's temperature. The wet-bulb thermometer shows a lower temperature,  $T_{wb}$ , than the dry-bulb thermometer,  $T_{db}$ , if the air is not saturated with water vapor. The temperatures are the same in saturated air. The amount of evaporation is dependent on the relative humidity of the air (more evaporation and lower  $T_{wb}$  with lower RH). Given the dry bulb and wet bulb temperatures the relative humidity is looked up on a psychrometric chart. For example, at sea level if  $T_{db} = 25\text{ }^{\circ}\text{C}$  and  $T_{wb} = 18\text{ }^{\circ}\text{C}$ ,  $\text{RH} = \sim 50\%$ . Looking up the %RH on a chart for every measurement is both time-consuming and cumbersome but can be automated with a microcontroller. A psychrometric sensor can achieve good precision with %RH resolutions of 0.01% for humidity ranges from 10–100% at temperatures from 0 to 60  $^{\circ}\text{C}$ , and accuracy of 1%. The disadvantages of a psychrometric sensor are a slow response time, large physical size, the need to keep one thermometer bulb wet, the need to have airflow around the wet bulb, and high cost.

### 7.2. Chilled Mirror Hygrometers

The chilled mirror hygrometer is considered the most accurate and reliable hygrometer. Chilled mirror hygrometers use a cooled mirror with an optoelectronic mechanism to detect condensation on the mirror surface at an accurately measured temperature. The system is configured to reflect LED light off a mirror at an angle of approximately 45 degrees with a photo-transistor detecting the reflected light. The temperature of the mirror is electronically controlled, typically with a Peltier-effect device. The system works by cooling the mirror's surface below ambient temperature until condensation forms, causing the LED's light to scatter resulting in a sudden drop in the output of the photo-transistor. The surface temperature of the mirror is read using an accurate temperature sensor such as a thermistor. The temperature at which condensation forms is the dew point. All humidity values can be calculated from the dew point. The mirror temperature can be controlled with a feedback loop to continuously track the dew point. The chilled mirror is the most stable and accurate method to determine relative humidity. However, it is crucial to keep the mirror clean, provide a method of clearing the condensation and to ensure that the temperature sensor and mirror are of high quality. This method operates over the full humidity range (0-100%RH) and can be used for numerous gases at many pressures. Chilled mirror hygrometer instruments are bulky and very expensive.

### 7.3. Mechanical Hygrometers

Mechanical hygrometers use sensing elements relying on a mechanical property of the sensor varying with humidity. The most common example is the animal hair hygrometer, which uses a piece of animal hair kept under tension. As humidity increases the hair becomes more flexible and stretches. A strain gage monitors the displacement caused by the hair stretching with a change in the moisture content of the air. The output of the strain gage is directly proportional to the relative humidity and is usually indicated on a mechanical meter. Mechanical hygrometers are generally compact, light weight, reliable, and inexpensive. Accuracies, however are in the  $\pm 10\%$  range.

### 7.4. Electronic Humidity Sensors

Electronic humidity sensors typically use either a change in resistance or capacitance to measure humidity. These sensors have become a popular choice because technology advances have made them accurate, compact, stable and low power. A capacitive sensor consists of two electrodes, separated by a dielectric. As the water vapor in air increases or decreases, the sensor's dielectric constant changes producing a higher (or lower) capacitance measurement corresponding to the humidity level. A resistive sensor consists of two electrodes, separated by a conductive layer. As the humidity in the air increases (or decreases) the conductivity of the sensing layer changes, altering the resistance between the two electrodes. New techniques for producing thin films have made these types of sensors, accurate, stable, and easy to manufacture in large quantities. The choice of material assures fast response times with little hysteresis. The accuracy of electronic sensors is limited by their drift over time caused by wide variations in temperature and humidity or the presence of pollutants.

The Si70xx humidity sensors use a MiM capacitor as the reference and perform a high-accuracy 24-bit conversion using a sigma delta conversion approach. Each part is factory calibrated for capacitance to RH with a minimum offset and slope correction. Later members of the family include non-linearity correction and temperature compensation.

Silicon Laboratories humidity sensors use the capacitance change due to moisture absorption of a polyimide film to sense humidity. The polyimide film is deposited over a metal finger capacitor and exposed to the ambient via an opening in the package. The polyimide material and sensing capacitor has been selected for excellent stability. An optional expanded polytetrafluoroethylene (ePTFE) hydrophobic filter provides protection against dust and most liquids.

The polyimide film is thin ( $<5 \mu\text{m}$ ) and very responsive to humidity (response time of less than 10 seconds). The hydrophobic filter has little impact on the response time.

While the Si70xx sensors are largely conventional, mixed-signal CMOS integrated circuits, relative humidity sensors in general, and those based on capacitive sensing using polymeric dielectrics in particular, have unique application and use requirements that are not common to conventional (non-sensor) ICs. Chief among those are:

- The need to protect the sensor during board assembly, i.e., solder reflow, and the need to subsequently
- Rehydrate the sensor.
- The need to protect the sensor from damage or contamination during the product life-cycle.
- The impact of prolonged exposure to extremes of temperature and/or humidity and their potential effect on
- Sensor accuracy.
- The effects of humidity sensor “memory”.

Each of these items is discussed in more detail in the following sections.

## 8. Humidity Control for Thermal Comfort

### 8.1. General Considerations

Humans are sensitive to humid air, because the human body uses evaporative cooling as the primary mechanism to regulate temperature. When the relative humidity and dew point are high, the rate of perspiration evaporation from the skin decreases because the amount of water vapor in the air is already close to saturation. Because humans perceive the rate of heat transfer from the body, rather than temperature itself, we feel warmer when the relative humidity is high. Relative humidity is a useful indication of how hot the weather “feels” and is more intuitive and easier to measure than other quantifications of water vapor in the air. Air conditioners are designed to maintain between 40–60% relative humidity in the occupied space.

**Table 1. Human Reaction to Humidity\***

Dew Point Temp (°F)	RH at 90°F	Human Perception
> 75	> 62%	Extremely uncomfortable
70 to 74	52% to 60%	Quite uncomfortable
65 to 69	44% to 52%	Somewhat uncomfortable
60 to 64	37% to 46%	Comfortable but humid
55 to 59	31% to 41%	Comfortable
50 to 54	31% to 37%	Very comfortable
< 49	< 30%	A bit dry

**\*Note:** See Lawrence, Mark G., “The Relationship between Relative Humidity and the Dewpoint Temperature in Moist Air—A Simple Conversion and Applications”, American Meteorological Society February 2005.

Humans tend to react with discomfort to dew points >61 °F (16 °C). The body perspires and produces sweat to cool down. High relative humidity and consequently a high dew point prevent the evaporation of sweat and reduce evaporative cooling. The body may overheat, resulting in discomfort.

Lower dew points, <50 °F (10 °C), generally correlate with lower ambient temperatures requiring the body to be less dependent on evaporative cooling. A lower dew point can be achieved with a high temperature at extremely low relative humidity allowing for effective cooling.

## 8.2. Heat Index

The heat index (HI), also called “apparent temperature” combines air temperature and relative humidity to estimate how “hot” a human will perceive the ambient conditions to be. The heat index is calculated for ambient temperatures above 27 °C (81 °F) and dew points above 12 °C (54 °F) (relative humidity above 40%). Heat index is calculated with the following equation<sup>2</sup>.

$$HI = -42.379 + (2.04901523T) + (10.14333127RH) - (0.22475541TRH) - (6.83783 \times 10^{-3}T^2) - (5.481717 \times 10^{-2}RH^2) + (1.22874 \times 10^{-3}T^2RH) + (8.5282 \times 10^{-4}TRH^2) - (1.99 \times 10^{-6}T^2RH^2)$$

Where:

HI = Heat Index in degrees Fahrenheit

T = Ambient temperature in degrees Fahrenheit

RH = Relative humidity in %

1. [www.shorstmeyer.com/wxfaq/humidity/humidity.html](http://www.shorstmeyer.com/wxfaq/humidity/humidity.html)
2. [www.srh.noaa.gov/images/epz/wxcalc/heatIndex.pdf](http://www.srh.noaa.gov/images/epz/wxcalc/heatIndex.pdf)
3. [www.crh.noaa.gov/pub/heat.php](http://www.crh.noaa.gov/pub/heat.php)

## 8.3. Wind Chill

Wind chill, also called wind chill temperature, wind chill factor or wind chill index, expresses the perceived air temperature on skin exposed to wind. Wind chill is defined for temperatures at or below 10 °C (50 °F) and wind speeds greater than 4.8 kilometers/hour (3.0 mph). Wind chill is calculated by the following equations<sup>4</sup>.

$$T_{wc} = 35.74 + 0.6215T_a - 35.75V^{0.16} + 0.4275T_aV^{0.16}$$

Where:

$T_{wc}$  = Wind chill temperature in degrees Fahrenheit

$T_a$  = Ambient temperature in degrees Fahrenheit

V = Wind velocity in miles-per-hour

**Table 2. Human Reaction to Heat Index, HI, in Shady Light Wind Conditions\***

Heat Index, HI, °F	Human Impact
80 to 90	Fatigue possible with prolonged exposure and physical activity
90 to 105	Sunstroke, heat cramps and heat exhaustion are possible
105 to 130	Sunstroke, heat cramps and heat exhaustion likely. Heat stroke possible
> 130	Heat stroke is highly likely with continued exposure

\*Note: See [www.crh.noaa.gov/pub/heat.php](http://www.crh.noaa.gov/pub/heat.php).

4. [http://web.uvic.ca/~eos340/wind\\_chill.pdf](http://web.uvic.ca/~eos340/wind_chill.pdf)

## 8.4. Thermal Stress

Heat index and wind chill are a measure of how the temperature is perceived by humans and can be referred to collectively as “apparent temperature” or “relative outdoor temperature”.

The human body loses heat by conduction and convection. The rate of heat loss by the body depends on the amount of exposed skin and the wind speed. The human body responds to heat loss by attempting to maintain its surface temperature. Rapid heat loss results in both the perception of lower temperatures and an actual greater heat loss as the body attempts to maintain body temperature on exposed skin increasing the risk of hypothermia, frostbite and death.

**Table 3. Wind Chill Temperature Chart\***

$VT_a$	40 °F	30 °F	20 °F	10 °F	0 °F	-10 °F	-20 °F	-30 °F
10 mph	34	21	9	-4	-16	-28	-41	-53
20 mph	30	17	4	-9	-22	-35	-48	-61
30 mph	28	15	1	-12	-26	-39	-53	-67
40 mph	27	13	-1	-15	-29	-43	-57	-71
50 mph	26	12	-3	-17	-31	-45	-60	-74

\*Note: The shaded area indicates a danger of frostbite.

## 8.5. Environmental Quality

Traditionally an environment was controlled based on a temperature measurement. In recent years the relative humidity measurement has become just as important. Humidity control is especially important in living, storage, and manufacturing sites. Control of temperature and relative humidity is also critical in the preservation of many materials including medication, food, fabric and wood products. Unacceptable levels of temperature and/or relative humidity contribute significantly to the breakdown of materials. Heat accelerates deterioration, whereas high relative humidity provides the moisture necessary, to promote harmful chemical reactions and, in combination with high temperature, encourages mould growth and insect activity. Extremely low relative humidity, which can occur in winter in centrally heated buildings, may lead to desiccation of some materials causing them to become brittle. Large fluctuations in temperature and relative humidity are damaging due to expansion and contraction which can accelerate deterioration. Installation and operation of adequate climate controls to meet preservation standards will retard the deterioration of materials considerably, while maintaining a comfortable environment for humans. For example, luthiers recommend keeping fine wooden musical stringed instruments such as violins, cellos and guitars at a RH between 45% and 55% and temperatures between 16 °C (55 °F) to 21 °C (70°F) to prevent warping and splits in the wood. The ideal temperature and relative humidity will vary depending on the material and application. Typically, a steady temperature of 16 to 21°C and a relative humidity between 30% and 60% suffices for many applications. Low power battery operated temperature and humidity sensors can be used to monitor conditions during shipment or storage for food and a variety of other materials.

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## APPENDIX A—INDUSTRY SPECIFICATIONS AND GUIDELINES

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- A.1 ANSI/ASHRAE Standard 55
- A.2 BS1339
- BS 1339-1:2002 Part 1: Terms, definitions and formulae
- BS 1339-2:2009 Part 2: Humidity calculation and tables - User guide
- BS 1339-3:2004 Part 3: Guide to the measurement of humidity
- U.K. National Physics Laboratory Guide to RH Measurement
- <http://www.npl.co.uk/publications/good-practice-online-modules/humidity/>

The following topics can be found on Wikipedia:

- Relative Humidity
- Dew Point
- $^{\circ}\text{C}$
- IP Rating
- Polymers
- Wave Soldering

**Si70xx certificate of compliance (with web link)**

## APPENDIX B—EQUATIONS FOR VAPOR PRESSURE AND HUMIDITY CALCULATIONS

Many equations have been developed to express humidity parameter relationships. They can generally be broken down into two groups, those developed from thermodynamic principles and equations empirically derived from experimental data. The Clapeyron and Clausius-Clapeyron equations will be presented as examples of the first group and the Sonntag, Magnus and Antoine equations as examples of empirically derived expressions. The empirically derived expressions are generally easier to use but have limitations to the range of use and accuracy. Approximations may be useful to further simplify calculations and conversions between humidity parameters as long as the limitations imposed by the approximations are fully comprehended. The use of these equations and approximations will be discussed below.

### Clapeyron Equation

The Clapeyron equation is based on one of the Maxwell thermodynamic relationships contains no approximations and provides an exact solution. It considers saturation pressure and temperature, the change of entropy associated with a change of phase and the change in volume of the two phases and represents the slope of the vapor-pressure curve. The Clapeyron equation can be expressed as follows:

$$\frac{\partial p'}{\partial T} = \frac{\Delta s}{\Delta v}$$

where:

$p'$  = Saturation Vapor Pressure

T = Temperature in °K

$\Delta s$  = Entropy Change between the two Phases

$\Delta v$  = Volume Change

since:

$$\Delta s = \frac{\Delta h}{T} \text{ Across a phase transition}$$

where:

$\Delta h$  = Enthalpy Change between the two Phases

The Clapeyron Equation can be rewritten as follows:

$$\frac{\partial p'}{\partial T} = \frac{\Delta h}{\Delta v T}$$

The Clapeyron equation is valid for all phase transitions (solid/liquid, solid/gas and liquid/gas) and represents the slope of the phase change boundaries. The parameters in this equation that can be directly measured are temperature, pressure and volume. Entropy and enthalpy can only be measured indirectly in terms of the other parameters.

## Clausius-Clapeyron Equation

The Clausius-Clapeyron equation modifies the Clapeyron equation with two simplifying approximations that make this equation useful for ice to water vapor and liquid water to water vapor transitions. The first assumption is that the change in volume from liquid water to gas (water vapor) or solid (ice) to gas (water vapor) is approximately equal to the volume of the gas (water vapor).

$$\Delta V = V_{\text{gas}} - V_{\text{liquid}} \approx V_{\text{gas}}$$

$$\Delta V = V_{\text{gas}} - V_{\text{solid}} \approx V_{\text{gas}}$$

The second approximation is the gas (water vapor) can be treated as an ideal gas.

$$V_{\text{gas}} = \frac{nR'T}{P}$$

Incorporating these assumptions in the Clapeyron equation yields the following.

$$\left(\frac{\partial p'}{\partial T}\right) = \frac{\Delta h}{T \frac{nR'T}{p'}} = \frac{\Delta h}{RT^2 \frac{1}{p'}}$$

Rearranging:

$$\left(\frac{dp'}{p'}\right) = \frac{\Delta h}{RT^2} dT$$

Integrating as an indefinite integral and assuming  $\Delta h$  is constant:

$$\int \frac{dp'}{p'} = \frac{\Delta h}{R} \int \frac{1}{T^2} dT$$

$$\ln(p') + C_0 = \frac{-\Delta h}{R} \left(\frac{1}{T}\right) + C_1$$

Where:

$\Delta h$  = enthalpy change for phase change which varies between  $2.501 \times 10^6$  and  $2.257 \times 10^6$  J/kg in the range of 0–100 °C

$R'$  = universal gas constant

$R$  = specific gas constant for water which is 461.5 J/(K – kg)

Rearranging:

$$p' = C_2 e^{\frac{-\Delta h}{R} \left(\frac{1}{T}\right)}$$

Accuracy is best around the temperature used to calculate  $C_2$ .

For example  $C_2 = 2.53 \times 10^{11}$  Pa at 0 °C

## Humidity-Related Calculations

While the above expressions are physically based, they are difficult to solve and manipulate. Hence, many approximate formulas have been developed. The two most common are the Antoine Equation and the Magnus equation. Many of these formulas contain coefficients that can vary depending on their source, its age and, in some cases, the context of the equations use.

### Antoine Equation

This equation calculates saturation vapor pressure. Additionally; coefficients are available for this equation for a wide variety of vapors other than water. The coefficients used in the following equation are for an air-water system and are optimized for use over the temperature range 0 to 100 °C.

$$p_s = \exp\left(23.19 - \frac{3830}{T - 44.83}\right)$$

$$t_{dp} = \frac{3830}{23.19 - \ln(P_s)} + 44.83$$

Where:

Pressure in Pascals ( $\text{Nm}^{-2}$ )

Temperature in °K ( $^{\circ}\text{C} + 273.15$ )

### Magnus Equation

This equation calculates saturation vapor pressure as does the Antoine Equation. It has the advantage that it can be easily manipulated to find the dew point ( $t_{dp}$ ) or frost point.

$$P'_s = C_1 e^{\left(\frac{A_1 T}{B_1 + T}\right)}$$

Where:

$P'_s$  is pressure in Pascals  $\text{Nm}^{-2}$

T is temperature in °C

Over the range of -40 to +50 °C, the best fit constants are as follows:

$A_1$	$B_1$	$C_1$	Reference
17.625	243.04	610.94	Alduchov, Oleg A., Robert E. Eskridge, 1996: Improved Magnus Form Approximation of Saturation Vapor Pressure. <i>J. Appl. Meteor.</i> , <b>35</b> , 601–609

For air with a vapor pressure  $P'$ , the dew point is defined as the temperature at which the water vapor would be saturated. Thus,

$$T_d = B_1 \times \frac{\ln\left(\frac{P'}{C_1}\right)}{A_1 - \ln\left(\frac{P'}{C_1}\right)}$$

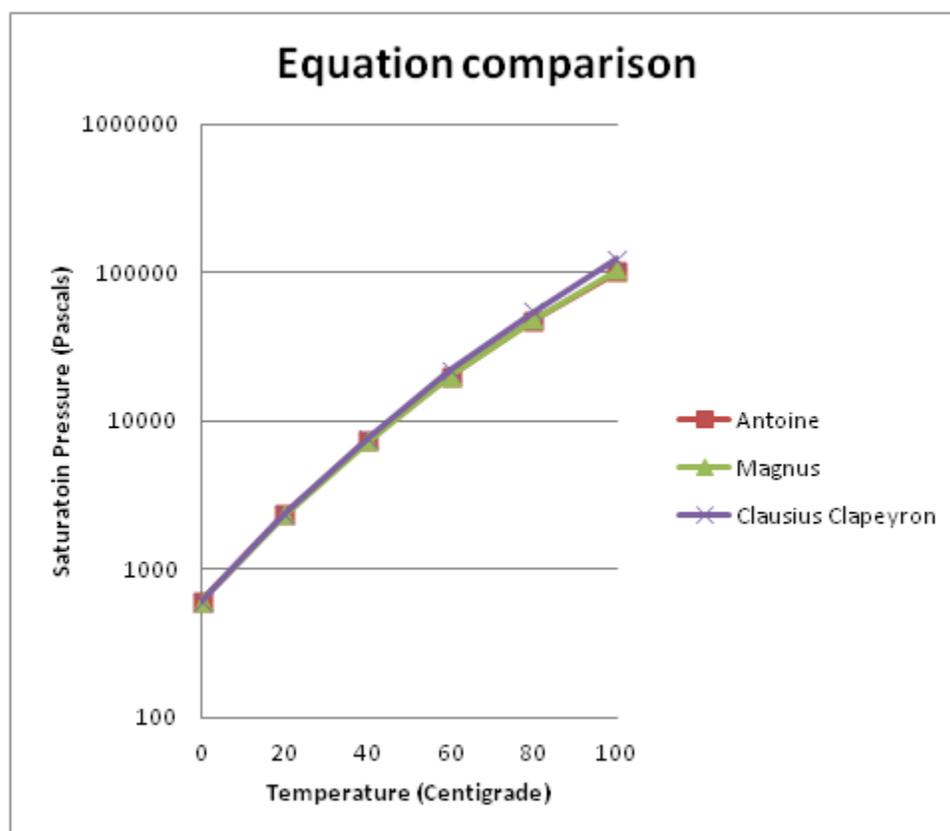
Utilizing the relationship

$$\text{RH} = 100 \times P'/P'_s$$

$$T_d = \frac{B_1 \left[ \ln\left(\frac{RH}{100}\right) + \frac{A_1 T}{B_1 + T} \right]}{A_1 - \ln\left(\frac{RH}{100}\right) - \frac{A_1 T}{B_1 + T}}$$

## Equation Comparison

The following graph compares the above equations:



## Mark Lawrence Rule of Thumb\*

This easy to use equation provides an estimate of change in dew point for a change in relative humidity or can be easily reversed to estimate a change in relative humidity for a known change in dew point. This approximation is valid for RH>50%. This rule of thumb says that the dew point temperature decreases approximately 1C for every 5% decrease in RH starting at tdp = t and RH=100%. This relationship is very handy if very little computational capability is available and the accuracy limitations and range of applicability are acceptable.

**\*Note:** See Lawrence, Mark G., "The Relationship between Relative Humidity and the Dewpoint Temperature in Moist Air - A Simple Conversion and Applications", American Meterological Society February 2005.

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## Estimating RH with Heating

### Equation Development

The Magnus equation for partial pressure of water in air is:

$$P = \left(\frac{RH}{100}\right) \times C_1 e^{A_1 \times \frac{T}{(B_1+T)}}$$

for a given relative humidity RH in percent and temperature in °C. Typical values for A1, B1, and C1 are 17.625, 243.04, and 610.94, respectively.

If the air is heated, the partial pressure does not change, and the apparent relative humidity drops according to:

$$P = \left(\frac{RH'}{100}\right) \times C_1 e^{A_1 \times \frac{(T+\Delta T)}{(B_1+T+\Delta T)}}$$

or

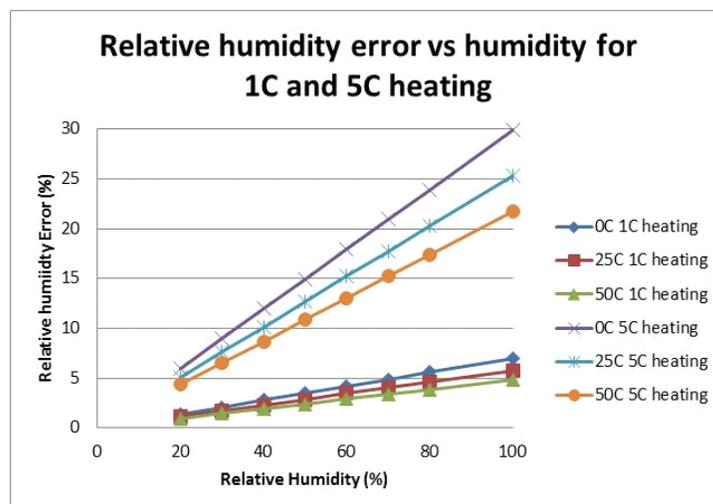
$$RH' = RH \frac{e^{\frac{A_1 \times T}{(B_1+T)}}}{e^{\frac{A_1 \times (T+\Delta T)}{(B_1+T+\Delta T)}}$$

this can be simplified to:

$$RH' = RH e^{\frac{-A_1 \times B_1 \times T}{(B_1+T)(B_1+T+\Delta T)}}$$

## Linearization

The above equation is still too complex to be useful in simple systems; however, it can be noted that, over a narrow temperature range, the relative humidity error is fairly linear with relative humidity for a given amount of heating and ambient temperature.



The error is linear with RH and increases by about 5% RH per °C (this is the familiar “Mark Lawrence Rule of Thumb”).

Thus, the actual RH can be estimated fairly accurately by:

$$RH = \frac{RH_{\text{measured}}}{(1 - 0.05\Delta T)}$$

The accuracy of this estimate can be improved by measuring the temperature of the RH sensor and correcting for heating to get the ambient temperature.

$$T_{\text{ambient}} = T_{\text{measured}} - \Delta T$$

For 5 °C heating, the correction factor varies from 0.0598 at 0 °C to 0.0435 at 50 °C ambient or

$$CF = 0.0598 - 0.000346 \times T_{\text{ambient}}$$

and finally,

$$RH = \frac{RH_{\text{measured}}}{(1 - CF \times \Delta T)}$$

## APPENDIX C—TERM, UNIT, AND COEFFICIENT REFERENCE

- **Absolute vapor pressure**—A measure of the actual amount of water present in the air.
- **Dew Point**—For a given RH and temperature, the temperature at which condensation would form if the air were cooled; meaningful as an indicator of comfort.
- **Hydrophobic**—Water repellent/resistant.
- **IP Rating**—Ingress Protection Rating; first digit indicates level of protection against particle; the second digit represents level of protection against liquids.
- **IP67**—An ingress protection rating indicating that the assembly is dust tight (6) and can withstand up to 1 m of water pressure (7).
- **Kapton**—A polyimide film developed by DuPont that is stable over a wide temperature range (up to ? +400 °C). It is available in sheet, tape, and “dot” form and is used to protect selected components during solder reflow.
- **Oleophobic**—Oil repellent/resistant.
- **Relative Humidity**— $\text{Absolute\_Vapor\_Pressure} \div \text{Saturated\_Vapor\_Pressure}$ ; expressed as a percentage.
- **Saturated vapor pressure**—The maximum amount of water that the air can hold; dependent on temperature.

**Table 4. Common Pressure Unit Conversions**

		Divide to Convert						
	Unit	Pascal (Pa)	Hectopascal (hPa)	Kilopascal (kPa)	Bar (bar)	Atmosphere (atm)	Torr* (torr)	lbf/in <sup>2</sup> (psi)
<b>Multiply to Convert</b>	Pa	1	1x10 <sup>-2</sup>	1x10 <sup>-3</sup>	1x10 <sup>-5</sup>	9.8692x10 <sup>-6</sup>	7.5006 x 10 <sup>-3</sup>	145.04 x 10 <sup>-6</sup>
	hPa	100	1	1x10 <sup>-1</sup>	1x10 <sup>-3</sup>	9.8692x10 <sup>-4</sup>	7.5006 x 10 <sup>-1</sup>	145.04 x 10 <sup>-4</sup>
	kPa	1000	10	1	1x10 <sup>-2</sup>	9.8692x10 <sup>-3</sup>	7.5006	145.04 x 10 <sup>-3</sup>
	bar	100,000	1000	100	1	0.98692	750.06	14.50377
	atm	101,325	1013.25	101.325	1.01325	1	760	14.696
	torr	133.322	1.33322	133.322 x 10 <sup>-3</sup>	1.3332 x 10 <sup>-3</sup>	1.3158 x 10 <sup>-3</sup>	1	19.337 x 10 <sup>-3</sup>
	psi	6.895 x 10 <sup>3</sup>	68.95	6.895	68.948 x 10 <sup>-3</sup>	68.046 x 10 <sup>-3</sup>	51.715	1

**\*Note:** 1 torr ≈ 1 mmHg

**Temperature Conversions:**

$$T_{\circ C} = \frac{5}{9}(T_{\circ F} - 32)$$

$$T_{\circ F} = \left(\frac{9}{5}\right)T_{\circ C} + 32$$

$$T_{\circ K} = T_{\circ C} + 273.15$$

$$T_{\circ R} = T_{\circ F} + 459.69$$

Table 5. Humidity Terms and Definitions

Term	Definition	Units
$p$	Pure water vapor pressure (no air or other gas)	$N/m^2$ or Pa
$p'$	Actual vapor pressure (water vapor in air)	$N/m^2$ or Pa
$p_s$	Pure water saturated vapor pressure (no air or other gas)	$N/m^2$ or Pa
$p_s'$	Actual saturated vapor pressure (water vapor in air)	$N/m^2$ or Pa
$p$	Total atmospheric pressure	$N/m^2$ or Pa
$f$	Water vapor enhancement factor	dimensionless
$d_v$	Volumetric humidity mass of water vapor/volume of humid gas	$kg/m^3$
$M_g$	Molal mass (molecular weight) of dry gas (air)	kg/mol
$M_v$	Molal mass (molecular weight) of water vapor	kg/mol
$m$	mass	kg
$n$	Amount of substance in moles	mol
$R$	Universal gas constant in joules per mole of air per K	J/(mol)(K)
$R'$	Universal gas constant in joules per kilogram of air per K	J/(kg)(K)
$S$	Percent of saturation	%
$\Psi$	% Relative humidity (%RH or %rh)	%
$Y$	Mixing ratio kg vapor/kg dry gas	kg/kg
$Y_w$	Specific humidity kg vapor/kg humid gas	kg/kg
$\Delta h$	Enthalpy change between phases $h_{ig}$ Ice $\rightarrow$ gas $h_{if}$ Ice $\rightarrow$ liquid $h_{fg}$ Liquid $\rightarrow$ gas	
$\Delta s$	Entropy change between phases	
$t_{dp}$	Dewpoint temperature	

## APPENDIX D—NONLINEAR CORRECTION OF VOLTAGE INPUTS WITH THE Si7013

The Si7013 has the capability to apply a lookup-table-based non-linear correction to voltage measurements. This correction is invoked by writing a “1” to bit 5 of user register 1. Note that humidity measurements should not be performed when this bit is set. In the discussion below, “input” refers to the A/D voltage measurement result, which is a 16-bit signed integer, and “output” refers the output after the non-linear correction, which is assumed to be a 16-bit unsigned integer.

The non-linear correction is based on a 10-point table lookup linearization. Each point consists of the ideal output for a given expected A/D measurement result. Table 6 is stored in the Si7013 memory, which must also have the slope at points 1–9. Slope is multiplied by a scaler of 256.

$$\text{slopeN} = 256 \times \frac{(\text{outputN} + 1 - \text{outputN})}{(\text{inputN} + 1 - \text{inputN})}$$

Only nine of the input/output pairs need to be in the table because the 10th pair is determined by the slope equation. Overall, the Si7013 has 27 16-bit numbers in its table (54 bytes). This table is stored in non-volatile memory of the Si7013 and must be programmed based on the desired look-up table.

The actual output is determined by extrapolation:

If in > in2, out = out1 + slope1 x (in - in1) / 256

Else if in > in3, out = out2 + slope2 x (in - in2) / 256

Else if in > in4, out = out3 + slope3 x (in - in3) / 256

Else if in > in5, out = out4 + slope4 x (in - in4) / 256

Else if in > in6, out = out5 + slope5 x (in - in5) / 256

Else if in > in7, out = out6 + slope6 x (in - in6) / 256

Else if in > in8, out = out7 + slope7 x (in - in7) / 256

Else if in > in9, out = out8 + slope8 x (in - in8) / 256

Else out = out9 + slope9 x (in - in9) / 256

The table must be arranged in order of decreasing input values. The table is entered into memory addresses 0x82–0xB7 one byte at a time.

**Table 6. Memory Location Descriptions**

Memory Location	Description
0x82	MSB of in1
0x83	LSB of in1
...	...
0x93	LSB of in9
0x94	MSB of out1
...	...
0xA5	LSB of out9
0xA6	MSB of slope1
...	...
0xB7	LSB of slope9

The table itself is user-programmed, and, by default, all table values are 0xFF. It should be noted that, once the non-linear correction data is saved to memory, it cannot be overwritten.

As an aid to calculation of the table, several tools have been developed.

- A spreadsheet, “linearization.xlsx”, is available. This spreadsheet calculates the expected output of the A/D based on an assumed thermistor and biasing circuit (the NCP18XH103F03RB thermistor used on Si7013 evaluation boards with 24.3 kΩ biasing resistors). Then, based on the desired output after linearization (In this case, output = (temperature + 46.85) x 65536/175.72), the slope is calculated. Finally, the spreadsheet calculates the hexadecimal values that should go in memory locations 0x82–0xB7 based on the input/output and slope values.
- The Si7013 evaluation board has the option of trying different values of linearization based on numbers entered in a GUI. These values can be saved to file or burned into the Si7013 memory. For example:

For the Si7013 evaluation board with a 10 kΩ thermistor and two 24.3 kΩ bias resistors, and assuming the A/D conversion is done using VDD as a reference with buffered inputs, the ideal input voltage version temperature is:

$$V_{in} = VDD \times R_{thermistor} / (R_{thermistor} + 46.4 \text{ k}\Omega)$$

Since VDD is also the reference, then the expected A/D conversion result is:

$$A/D \text{ counts} = 32768 \times R_{thermistor} / (R_{thermistor} + 46.4 \text{ k}\Omega)$$

If it is desired to linearize this result for the same temperature representation as the onboard temperature sensor:

$$\text{Temperature } ^\circ\text{C} = (\text{Output\_Code} \times 175.72 / 65536 - 46.85)$$

Then, the desired output code is:

$$\text{Output\_Code} = 65536 \times (\text{Temperature} + 46.85) / 175.72$$

Using thermistor data sheet values of resistance versus temperature and choosing to linearize at the points (–15 °C, –5 °C, 5 °C, 15 °C, 25 °C, 35 °C, 45 °C, 55 °C, 65 °C, and 75 °C) results in the information listed in Table 7.

**Table 7. Example Non-Linear Correction to Thermistor Voltage Measurements**

Thermistor Resistance (from Data Sheet)	Vin/Vdd Assuming 24.3 kΩ Bias Resistors	A/D Codes	Desired (Temperature) Code	Slope
53650	0.524694377	17193	11879	–256
33892	0.410851961	13463	15608	–294
22021	0.31181943	10218	19338	–364
14674	0.231912002	7599	23067	–476
10000	0.170648464	5592	26797	–640
6948	0.125081011	4099	30527	–877
4917	0.091877347	3011	34256	–1210
3535	0.067804738	2222	37986	–1684
2586	0.050521627	1655	41715	–2346

The values highlighted in gray would be the table entries for the Si7013.

## D.1 Entering the Table into Memory

The table is entered into memory addresses 0x82–0xB7, one byte at a time. For the above example, the values to be written are listed in Table 8:

**Table 8. Example Non-Linear Thermistor Correction Entries into Si7013 Memory**

Table Entry	Hex	Byte 1 Byte 2	Memory Location	Table Entry	Hex	Byte 1 Byte 2	Memory Location	Table Entry	Hex	Byte 1 Byte 2	Memory Location
17193	4329	43	82	11879	2E67	2E	94	-256	FF00	FF	A6
		29	83			67	95			00	A7
13463	3497	34	84	15608	3CF8	3C	96	-294	FEDA	FE	A8
		97	85			F8	97			DA	A9
10218	27EA	27	86	19338	4B8A	4B	98	-364	FE94	FE	AA
		EA	87			8A	99			94	AB
7599	1DAF	1D	88	23067	5A1B	5A	9A	-476	FE24	FE	AC
		AF	89			1B	9B			24	AD
5592	15D8	15	8A	26797	68AD	68	9C	-640	FD80	FD	AE
		D8	8B			AD	9D			80	AF
4099	1003	10	8C	30527	773F	77	9E	-877	FC93	FC	B0
		03	8D			3F	9F			93	B1
3011	0BC3	0B	8E	34256	85D0	85	A0	-1210	FB46	FB	B2
		C3	8F			D0	A1			46	B3
2222	08AE	08	90	37986	9462	94	A2	-1684	F96C	F9	B4
		AE	91			62	A3			6C	B5
1655	0677	06	92	41715	A2F3	A2	A4	-2346	F6D6	F6	B6
		77	93			F3	A5			D6	B7

The command code, 0xC5, is used for programming; so, for example, to program a Si7013 at slave address 0x40 with the values above starting with 0x4C to memory location 0x82, one would write:

```
<Start Condition> 0x40 W ACK 0xC5 ACK 0x82 ACK 0x2E ACK <Stop Condition>
```

.....

```
<Start Condition> 0x40 W ACK 0xC5 ACK 0xB7 ACK 0xD6 ACK <Stop Condition>
```

After writing all values, the chip should be reset by a power cycle or I<sup>2</sup>C reset command before attempting to use the correction mode of the voltage-sensing circuit.

It is not possible to change the values once written. However, to verify the values were written properly, use command 0x84. In the above example, to verify that 0x2E was written to location 0x82, use:

```
<Start Condition> 0x40 W ACK 0x84 ACK 0x82 ACK <Start Condition> 0x40R ACK 0x2E NACK <Stop Condition>
```

where 0x2E is the expected return value of the read transaction.

## D.2 How to Choose Linearization Points

The table must be arranged in order of decreasing values for the input. The slope can be positive or negative, and it is assumed that only positive output values are desired.

Generally, the input/output pairs should be chosen around the area where greatest accuracy is desired. If the transfer function is highly nonlinear and the more closely spaced the points, the less error there will be from interpolation. However, if accuracy is desired over a wide range, the input/output pairs should be spaced evenly over the range of interest. The spreadsheet allows estimation of errors from the extrapolation process.

## D.3 Other Register Settings

The voltage measurement configuration is determined by user register 2 of the Si7013. For thermistor measurement, it is assumed that the A/D input buffers will be used and that the A/D reference is VDD. The buffers are needed since the thermistor impedance is fairly large. VDD is used as a reference because the thermistor is biased relative to VDD. Thus, for a “hold master” measurement (SCL is held low during the measurement phase), Register 2 would be set to the following:

D7	D6	D5	D4	D3	D2	D1	D0
0	0	1	0	0	1	1	0/1
Not Used	Hold Master	Enable Linearization	7 msec Conversion Time	Not Used	Enable Buffer	VDD is Reference	Switch High or Low to Bias the Thermistor

## D.4 Noise and AC Pick Up

The A/D of the Si7013 is a delta sigma type converter, and the input is not sampled. Thus, it is assumed that the input voltage is constant over the measurement period. Generally, some amount of analog filtering prior to the A/D input is desirable. In the standard application circuit, this is accomplished with 0.1  $\mu$ F capacitors. These capacitors will form a filter at about 30 Hz, which is adequate for high-frequency noise pick up (e.g. am radio signals) but not for 60 Hz. If 60 Hz filtering is desired, these capacitors can be increased to >1  $\mu$ F, or the result could be digitally filtered (average of several measurements). If the sampling can be synchronized to 120 Hz (use the faster conversion time for this), then an average of just two samples would reject 60 Hz.

In the standard biasing circuit, the bias can be turned off between measurements to save power (change bit zero of user register 2). If this is done, allow adequate settling time between enabling the bias and making the measurement (use approximately 100 msec for the 0.1  $\mu$ F filter, which has a time constant of 30 msec).

## DOCUMENT CHANGE LIST

### Revision 0.1 to Revision 1.0

- Updated storage, handling, and assembly instructions.
- Corrected Table 4, “Common Pressure Unit Conversions,” on page 28.

### Revision 1.0 to Revision 1.1

- Multiple updates to include Si7013, Si7020, and Si7021 parts.

NOTES:

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